Continuum Mechanics 2016: Solutions to Exercise Set 1

Exercise 1: (Linear elasticity from harmonic oscillators)

Solution:

(a) The new spring constant k_N is defined implicitly through

$$F = k_N (L - L_{\rm eq}). \tag{1}$$

We have $\ell - \ell_{eq} = F/k$ and $N = L_{eq}/\ell_{eq} = L/\ell$, so that

$$L - L_{\rm eq} = N(\ell - \ell_{\rm eq}) = \frac{NF}{k}.$$
(2)

Upon comparing eqs. (1) and (2), we find

$$k_N = \frac{k}{N}.$$
(3)

(b) We denote the displacements $u_i \stackrel{\text{def}}{=} u(x_i, t)$. Newton's second law, where f_i denotes the sum of forces on particle *i*, yields

$$m\ddot{u}_i = f_i = k\left((u_{i+1} - u_i) - (u_i - u_{i-1})\right).$$
(4)

Thus,

$$\ddot{u}(x_i,t) = \frac{k}{m} \left(u(x_{i+1},t) - 2u(x_i,t) - u(x_{i-1},t) \right).$$
(5)

(c) We have $x_{i\pm 1} = x_i \pm \ell_{eq}$. Taylor expansion of $u(x_{i\pm 1}, t)$ yields

$$u(x_i \pm \ell_{\rm eq}, t) = u(x_i, t) \pm \ell_{\rm eq} \frac{\partial u(x, t)}{\partial x} \bigg|_{x=x_i} + \frac{\ell_{\rm eq}^2}{2} \frac{\partial^2 u(x, t)}{\partial x^2} \bigg|_{x=x_i} + \mathcal{O}(\ell_{\rm eq}^3).$$
(6)

Now, since this holds $\forall x_i$, we let $x_i \to x$, and since x is now also a continuous variable, the time derivative in eq. (5) becomes partial. Inserting eq. (6) in eq. (5) and omitting the $\mathcal{O}(\ell_{eq}^3)$ terms since $\ell_{eq} \ll 1$, we obtain

$$\frac{\partial^2 u}{\partial t^2} = \frac{k\ell_{\rm eq}^2}{m} \frac{\partial^2 u}{\partial x^2}.$$
(7)

(d) This the wave equation with the wave velocity $v = \ell_{eq}\sqrt{k/m}$. Realize this by inserting the general solution $u(x,t) = \tilde{u}(x \pm vt)$.

Exercise 2: (6.5 from Lautrup [1]) Find the eigenvalues λ_i and eigenvectors \mathbf{v}_i of

$$\boldsymbol{\sigma} = \begin{bmatrix} \boldsymbol{\tau} & \boldsymbol{\tau} & \boldsymbol{\tau} \\ \boldsymbol{\tau} & \boldsymbol{\tau} & \boldsymbol{\tau} \\ \boldsymbol{\tau} & \boldsymbol{\tau} & \boldsymbol{\tau} \end{bmatrix}.$$
 (8)

Solution: Straightforward calculation gives the eigenvalues $\lambda_1 = \lambda_2 = 0$ and $\lambda_3 = 3\tau$, and the corresponding eigenvectors $\mathbf{v}_1 = [1, -1, 0]$, $\mathbf{v}_2 = [1, 0, -1]$ (or any linear combination) and $\mathbf{v}_3 = [1, 1, 1]$. Normalized eigenvectors (as given in the book) are obtained by $\hat{\mathbf{v}}_i = \mathbf{v}_i/|\mathbf{v}_i|$.

Exercise 3: (6.6 from Lautrup [1]) Alternative formulation: Show that a stress tensor that is diagonal in all coordinate systems, have identical nonzero entries.

Solution: Let the stress tensor be given by $\boldsymbol{\sigma} = \text{diag}(\sigma_x, \sigma_y, \sigma_z)$ in the initial coordinate system. Consider a rotation by an angle θ about some fixed axis, and assume first that this is the z-axis. The associated rotation matrix is given by

$$\mathbf{R} = \begin{bmatrix} \cos\theta & -\sin\theta & 0\\ \sin\theta & \cos\theta & 0\\ 0 & 0 & 1 \end{bmatrix}.$$
 (9)

The stress tensor in the rotated coordinate system is given by

 $\sigma' = \mathbf{R} \sigma \mathbf{R}^{\top}$

$$= \begin{bmatrix} \sigma_x \cos^2 \theta + \sigma_y \sin^2 \theta & (\sigma_x - \sigma_y) \sin \theta \cos \theta & 0\\ (\sigma_x - \sigma_y) \sin \theta \cos \theta & \sigma_x \sin^2 \theta + \sigma_y \cos^2 \theta & 0 \end{bmatrix}$$
(11)

0

This should also be diagonal, $\sigma' = \text{diag}(\sigma_{x'}, \sigma_{y'}, \sigma_{z'})$. Therefore the off-diagonal entries must be zero $\forall \theta$, and so $\sigma_x = \sigma_y$. By symmetry, this argument applies to rotation about both the x- and y-axes as well, and thus $\sigma_x = \sigma_y = \sigma_z = \text{tr}(\sigma)/3 = -p$.

Exercise 4: (6.8 from Lautrup [1])

(a) Show that the average of a unit vector **n** over all directions obeys

0

$$\langle n_i n_j \rangle = \frac{1}{3} \delta_{ij}. \tag{12}$$

 σ_z

(b) Use this to show that the average of the normal stress acting on a surface element is (minus) the mechanical pressure.

Solution:

(a) • Method 1: Brute force it. A unit vector $\mathbf{n} = \mathbf{n}(\theta, \phi)$ is expressed by

$$\mathbf{n} = \begin{bmatrix} \sin\theta\cos\phi\\ \sin\theta\sin\phi\\ \cos\theta \end{bmatrix}.$$
 (13)

Hence,

$$\mathbf{n} \otimes \mathbf{n} = \begin{bmatrix} \sin^2 \theta \cos^2 \phi & \text{(sym.)} \\ \sin^2 \theta \sin \phi \cos \phi & \sin^2 \theta \sin^2 \phi \\ \sin \theta \cos \theta \cos \phi & \sin \theta \cos \phi \cos \phi & \cos^2 \theta \end{bmatrix}.$$
(14)

Carrying out all six integrals (straightforward, but cumbersome!) over all angles (θ, ϕ) and dividing by 4π , we get $\langle \mathbf{n} \otimes \mathbf{n} \rangle = \frac{1}{3}\mathbf{I}$, or eq. (12).

• Method 2: Use the insight from Exercise 3. The *average* of a quantity *over all directions* can not itself depend on the direction you evaluate it in. Using our acquired insight, we can write it on the form

$$\langle n_i n_j \rangle = k \delta_{ij},\tag{15}$$

where k is an undetermined constant. Taking the trace of both sides yields

$$3k = \langle n_i n_i \rangle = 1 \quad \Longrightarrow \quad k = \frac{1}{3}, \tag{16}$$

which gives eq. (12).

(b) The normal force on a surface element is given by the traction at the surface, $T_i^{(\mathbf{n})} = \sigma_{ij}n_j$, projected in the normal direction: $f^{(\mathbf{n})} = n_i T_i^{(\mathbf{n})} = n_i \sigma_{ij} n_j$. Taking the average,

$$\left\langle f^{(\mathbf{n})} \right\rangle = \left\langle n_i \sigma_{ij} n_j \right\rangle = \left\langle n_i n_j \right\rangle \sigma_{ij} = \frac{1}{3} \delta_{ij} \sigma_{ij} = \frac{\sigma_{ii}}{3} = -p.$$
 (17)

Here we have used that σ_{ij} is constant, and the definition of the (mechanical) pressure, $p = -\text{tr}(\boldsymbol{\sigma})/3 = -\sigma_{ii}/3$.

 B. Lautrup. Physics of Continuous Matter: Exotic and Everyday Phenomena in the Macroscopic World. CRC Press, second edition, 2011.